BSA Pre-Unit 4 Speed Transmission Ratios

Background & History

The 1st use of the transmission usually referred to generically as the "A10" is in the BB34 Goldstar with duplex cradle frame in 1953, with the remainder of the line following in 1954. These largest BSA machines ("A" Series 500 & 650cc twins and "B" Series 350 & 500cc single-cylinder) were originally offered with many different internal transmission ratio gear-sets to best equip the machines for different purposes such as normal road and moderate performance use, road race, and trials, sidecar and military & police, as shown below. Most of these parts are still available, and can be completely or partially retro-fit into transmission cases, both unit and pre-unit, back to . The exact gear tooth counts and ratios used in these gear-sets have varied slightly over time, and complete interchange may not be possible. Some of these parts were probably used in the older M20, M21, B31 &c. transmissions.

My purpose is to supply information about what choices are available based on combinations of factory parts, explain some of the differences between the various ratios, describe in a general way what needs to be done for a selection, and warn you of potential difficulties. I will limit my comments to methods that have either already been used successfully, or that I believe are sound concepts that are worthy of your time and effort. Ratio Calculations

BSA 4 speed transmission's intermediate gears (1st through 3rd) are the result of power transmission through two pairs of mating gears, one of which is the mainshaft high gear being turned by the layshaft (countershaft) high gear. The gear ratio is determined by taking the product of multiplying the two pair's tooth count ratios together: the high gear ratio times the ratio of the individual gear pair selected (1st, 2nd or 3rd).

Please note that the total tooth count of each pair of mating gears (mainshaft and layshaft) is 43: 26/17, 25/18, etc. This is fixed by the tooth form's pitch diameter and the center-to-center distance of each pair (which is obviously the distance between the shafts) is a constant. This is a useful safety check to see if the assembly you have is really a useable gear set, or merely a random assortment of parts.

Under certain conditions, a mating pair (with the 43 total) may be substituted for another ratio: 2nd for 3rd, etc. but this must be researched on an empirical basis and may require minor modifications to the length, dog shape and size, etc.

The high gear ratio is the ratio of the tooth counts of the high gear pair for the transmission type selected (see "High Gear Pair" in Column 4 below). For example, the STD. transmission (the most common model) has 27 teeth on the mainshaft high gear and 16 teeth on the layshaft gear, so the high gear multiplier (Column 5) will be 1.68750 for this transmission.

The 2nd number is the ratio of the tooth counts of the pair of gears (mainshaft & layshaft) for that ratio: 1st, 2nd or 3rd.

The std. transmission has 27 teeth on the mainshaft 1st gear and 16 teeth on the layshaft 1st gear, so the gear ratio for the 1st gear pair (Column 8) is 1.6875 for 1st gear in the std. transmission. To get the actual effective ratio of 1st gear, multiply those two numbers together: In 4th gear the mainshaft high gear is locked to the mainshaft, so the ratio is 1-1. No power is transmitted through the intermediate gears. Although they still turn, they're only idling.

This Table shows some common transmissions, their internal gear-sets, and how the individual ratios are generated.

For transmissions based on the Std (standard) high-gear pair, the high gear tooth counts are: 26 for the mainshaft gear and 17 for the layshaft gear, so the 1st number is always 1.54941 for transmission gear-sets based on the "std." high gear pair.

For transmissions based on the Close high-gear pair, the high gear tooth counts are: 25 for the mainshaft gear and 18 for the layshaft gear; the 1st number is always 1.38889 for transmission gear-sets based on the "close" high gear pair.

The overall governing component for all ratio selection is the high gear pair, which determines the range of ratio spread between 1st gear and 4th gear.

The two original factory gear-sets are shown in color for clarity: "Std." ratios in blue, and "Close" ratios in green.

Note that not all possible gear pair combinations were used in the original ratios shown in Table 8. I have inserted the other four theoretical choices, which may be possible (although I have no experience or information). Not all of them have any practical use.

Alternate 1st gear #1 (1.9319-1) creates an extremely close ratio, approximately the same ratio as the close ratio "RR., RR.T" (1.9290-1) and only suitable for road racing or LSR use. However, it allows this close ratio to be inserted in transmissions fitted with the std. ratio (26/17) high gear pair such as "DAY., DAY.T".

Alternate 2nd gear #2 (1.9319-1) creates a std. ratio in between the fairly wide "STD., STD.T" (1.7588-1) and the extremely wide "TRI., TRI.T" (2.3391-1) ratios in these std. ratio transmissions.

Alternate 2nd gear #3 (1.5972-1) creates a close ratio in between the fairly wide "SC., SC.T" and "ARRT., ASC., ACS.T, STD.2" (1.7545-1) and "RR.T2" and "RR., RR.T" (1.3257-1) ratios in these close ratio transmissions.

25/18

26/17

25/18

M/S

Gear

25

25

27

1.38889

1.54941

1.38889

19

18

18

16

Pair

Ratio

1.2632

1.3889

1.3889

1.6875

High Gear Multiplier		×	1 st Pair Ratio	=	1st Gear Ra	tio		
	1.54941	×	1.68750	=	2.58088-	1		
	Tabl	e 1	: Internal Gea	rbo	ox Ratios &	Gear Tooth	n Counts	;
Gea r	Ratio Type & Ca Marking	se	Actual Ratio]	High Gear Pair	Multiply By	L/S Gear	
1st	RR.T2		1.7545-1		25/18	1.38889	24	

1.9290-1

2.1242-1

2.3438-1

Alternate 3rd gear #4 (1.0000-1) has no practical purpose.

RR., RR.T

SC., SC.T

DAY., DAY.T

	STD., STD.T	2.5809-1	26/17	1.54941	27	16	1.6875		
	ARRT., ASC., ACS.T, STD.2	2.8770-1	25/18	1.38889	29	14	2.0714		
	TRI., TRI.T	3.1680-1	26/17	1.54941	29	14	2.0714		
	Alternate 1st #1	1.9319-1	26/17	1.54941	24	19	1.2632		
2nd	RR.T2	1.3257-1	25/18	1.38889	21	22	0.9545		
	RR., RR.T	1.3257-1	25/18	1.38889	21	22	0.9545		
	DAY., DAY.T	1.4598-1	26/17	1.54941	21	22	0.9545		
	SC., SC.T	1.7545-1	25/18	1.38889	24	19	1.2632		
	STD., STD.T	1.7588-1	26/17	1.54941	23	20	1.1500		
	ARRT., ASC., ACS.T, STD.2	1.7545-1	25/18	1.38889	24	19	1.2632		
	TRI., TRI.T	2.3391-1	26/17	1.54941	26	17	1.5494		
	Alternate 2nd #2	1.9319-1	26/17	1.54941	24	19	1.2632		
	Alternate 2nd #3	1.5972-1	25/18	1.38889	23	20	1.1500		
3rd	RR.T2	1.0996-1	25/18	1.38889	19	24	0.7917		
	RR., RR.T	1.0996-1	25/18	1.38889	19	24	0.7917		
	DAY., DAY.T	1.1012-1	26/17	1.54941	18	25	0.7200		
	SC., SC.T	1.3257-1	25/18	1.38889	21	22	0.9545		
	STD., STD.T	1.2108-1	26/17	1.54941	19	24	0.7917		
	ARRT., ASC., ACS.T, STD.2	1.3257-1	25/18	1.38889	21	22	0.9545		
	TRI., TRI.T	1.4598-1	26/17	1.54941	21	22	0.9545		
	Alternate 3rd #4	1.0000-1	25/18	1.38889	18	25	0.7200		
4th	RR.T2	1.0000-1	25/18	1.38889		NA			
	RR., RR.T	1.0000-1	25/18	1.38889					
	DAY., DAY.T	1.0000-1	26/17	1.54941					
	SC., SC.T	1.0000-1	25/18	1.38889					
	STD., STD.T	1.0000-1	26/17	1.54941					
	ARRT., ASC., ACS.T, STD.2	1.0000-1	25/18	1.38889					
	TRI., TRI.T	1.0000-1	26/17	1.54941					



				1				arec	.01	D/ II	mo		b
				/		PART	NUMB	ERS OF	GEAR	PINIO	NS & SI	AFTS	
						NUME	ER OF	TEETH	ON G	EAR PI	NIONS		
DESCRIPTION	GEARBOX MARKING	*	8	c	D	E	F	G	н	MAIN	LAY	SPEEDO	DRIVEN
EXTRA CLOSE ROAD RACING	RR.	67-3184 [25]	42-3022 [18]	67-3361 [22]	42-3026 [21]	67-3187 [24]	67-3198 [19]	67-3221 [18]	67-3216 [25]	67-3330	42-3019	42-3033	12-303
EXTRA CLOSE ROAD RACING	RR.T	67-3184 [25]	42-3086 [18]	67-3361	42-3026	67-3187	67-3198	67-3221	42-3087	67-3330	42-3094	42-3033	12-303
EXTRA CLOSE	RR.T2	42-3133	42-3086	67-3361	42-3026	42-3136	67-3198	42-3137	42-3138	42-3131	42-3094	42-3033	12-303
DAYTONA	DAY,	67-3207	12-3020	67-3361	42-3026	67-3223	(19)	[19]	[24] 67-3216	67-3330	42-3019	67-3088	. 67-317
DAYTONA	DAY.T	[26]	[17]	[22]	[21]	(25)	(18)	[18]	(25)	(2.2220	12-3025	61-3000	07-317
		[26]	[17]	[22]	(21)	[25]	(18	[18]	(25)	07-3330	42-3094	67-3088	67-317
SCRAMBLES	sc.	67-3311 [25]	*2-3022 [18]	67-3302 [19]	42-3024 [24]	67-3305 [22]	67-3212 [21]	67-3191 [16]	42-3097 [27]	67-3315	42-3019	42-3033	12-303
SCRANBLES	SC.T	42-3088 [25]	42-3086 [18]	67-3302 [19]	42-3024 [24]	67-3305 [22]	67-3212 [21]	67-3191 [16]	42-3210 [27]	67-3315	42-3094	42-3033	12-303
A10 SPITFIRE 1957 ONLY	SC.T2	42-3157 [25]	42-3086 [18]	67-3302 [19]	42-3024 [24]	67-3305 [22]	67-3212 [21]	67-3191 [16]	42-3210 [27]	42-3156	42-3094	42-3033	42-303
STANDARD	STD.	67-3192 [26]	42-3020 [17]	67-3201 [20]	42-3023	67-3202	67-3198	67-3191	42-3097	67-3330	42-3019	67-3088	67-317
STANDARD	STD.T	42-3076	42-3081	67-3201	42-3023	67-3202	67-3198	67-3191	42-3210	67-3330	42-3094	67-3088	67-317
TRIALS	TRI.	67-3309	42-3020	67-3301	42-3025	67-3210	67-3212	67-3313	67-3213	67-3313	42-3019	67-3088	67-317
TRIALS	TRI.T	42-3091	42-3081	67-3301	42-3025	67-3210	67-3212	67-3313	42-3093	67-3313	42-3094	67-3088	67-317
B34 GOLD STAR	ARRT.	12-3088	42-3086	67-3302	42-3024	67-3210	(21)	[14] 67-3313	[29] 42-3093	67-3313	42-3094	42-3033	42-303
B34 GOLD STAR	ASC.T	[25] 42-3088	18]	[19] 67-3302	[24]	[22]	[21]	[14]	(29)	67 2212	#2 200h	#2 2022	
CATALINA 1962	STD.2	[25]	[18]	(19)	[24] 42-3024	[22]	[21]	[14]	[29]	67 2222	12-3094	42-3033	42-303
ROCKET 1962	ARRT.	[25]	[18] 42-3086	(19)	[24]	(22)	(21)	(14)	[29]	67 2212	12-3019	42-3033	42-303
TWIN 1963		[25]	[18]	[19]	[24]	[22]	[21]	[14]	[29]	07-3313	42-3094	42-3033	42-303
SPITFIRE 1963	ASC.	42-3088 [25]	42-3022 [18]	67-3302 [19]	42-3024 [24]	67-3210 [22]	67-3212 [21]	67-3313 [14]	67-3213 [29]	67-3313	42-3019	42-3033	42-303
TO FIND AS FOLLO DIVIDE TI NUMBER	THE GEAR R WARD FOOTCH THE GEAR R WS : HE NUMBER OF TEETH (OF TEETH (ATTES NEE CHANGE CA ATTOS OF OF TEET ON THE EP	A MACHI H ON TH GINE SPR	ERS ON LA E IS PART N IS PART N NE CALCU IE CLUTCI OCKET AN	YSHAFT A No. 67-333 Io. 42-3011. JLATE THI H SPROCK	ND MAINS 2 E TOP GEA KET BY TH PLY BY TH PLY BY TH	HAFT. IR IE IE						
NUMBER	OF TEETH	N THE G	EARBOX	PROCKET	AS EXAM	IPLE :			G	ARBOXI	NTERNAL	RATIOS	
CLUTCH SPROCKET (4) R/W SPROCKET (42) 1006 X CH SPROCKET (41) - 5591							RR. C	or RR.T	1	1.099	L. 326	1.929	
TO FIND THE INTERMEDIATE GEAR RATIO MULTIPLY THE OVERALL TOP GEAR BY THE INTERNAL GEAR RATIO CONCERNED, AS EXAMPLE :								RR. 72	1	1	1.099	1.326	1.754
								DAY.	OF DAY.T	1	1.101	1.460	2.124
TOP GEAR 5.5H OR 5.6 X BOTTOM GEAR INTERNAL RATIO							SC. C	or SC.T	1	1.325	1.754	2.343	
13	=14.4 BOTT	UM GEAR	VERALL	RATIO.	INION C	EEVE		TRI.	or TRL.T	1	1.459	2.339	2.580
GEARBOX INTERNAL RATIO							ARRT. ASC 1					3. 201	